

# NON-SINGULAR METHOD OF FUNDAMENTAL SOLUTIONS FOR THE DEFORMATION OF TWO-DIMENSIONAL ELASTIC BODIES IN CONTACT

## NESINGULARNA METODA FUNDAMENTALNIH REŠITEV ZA DEFORMACIJO DVO-DIMENZIONALNIH STIKAJOČIH SE ELASTIČNIH TELES

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The development of an effective new numerical method for the simulation of the micromechanics of multi-grain systems in contact is developed in the present paper. The method is based on the Method of Fundamental Solutions (MFS) for two-dimensional plane strain isotropic elasticity and employs the Kelvin Fundamental Solution (FS). The main drawback of MFS is the presence of an artificial boundary, outside the physical boundary, for positioning the source points of the FS, which is difficult or impossible in multi-body problems. In order to remove the singularities of the FS the point sources are replaced by the distributed sources over circular disks. The values of the distributed sources are calculated in a closed form in the case of the Dirichlet boundary conditions. In the case of the Neumann boundary conditions the respective values of the derivatives of the FS are calculated indirectly from the considerations of the solution of simple displacement fields. A problem of two, four and nine bodies in contact is tackled. The newly developed method is verified based on a comparison with the classic MFS. The numerical method will form a part of the microstructure-deformation model, coupled with the macroscopic thermo-mechanics simulation system for continuous casting, hot rolling and heat treatment.

Keywords: isotropic elasticity, plane strain, Navier's equation, displacement and traction boundary conditions, non-singular method of fundamental solutions, Kelvin's fundamental solution

V članku opisujemo učinkovito novo numerično metodo za simulacijo mikromehanike sistemov z več zrn v stiku. Ta temelji na metodi fundamentalnih rešitev (MFR) za dvo-dimenzionalne probleme izotropnih elastičnih ravninskih deformacij in uporablja Kelvinovo fundamentalno rešitev (FR). Bistvena slabost MFR je prisotnost fiktivnega roba zunaj fizikalnega roba za postavitev izvornih točk FR, kar je težavno ali nemogoče pri problemih z več telesi. Z odstranitvijo nesingularnosti FR smo točkovne izvire nadomestili s porazdeljenimi izviri po krogih. Vrednosti porazdeljenih izvirov so izračunane v analitični obliki za Dirichletove robne pogoje. Pri Neumann-ovih robnih pogojih so vrednosti odvodov FR izračunane posredno pri upoštevanju rešitev preprostih polj premika. Obravnavani so sistemi z dvema, štirimi in devetimi telesi v stiku. Nova metoda je verificirana na podlagi primerjave s klasično MFR. Uporabljena bo v deformacijskem modelu mikrostrukture, povezanim z makroskopskim termomehanskim sistemom za simulacijo kontinuirnega ulivanja, vročega valjanja in toplotne obdelave.

Ključne besede: izotropna elastičnost, ravninska deformacija, Navierove enačbe, robni pogoji premika in vlečenja, nesingularna metoda fundamentalnih rešitev, Kelvinova fundamentalna rešitev

## 1 INTRODUCTION

The physical modelling of metallurgical processes<sup>1</sup> consists of modelling the relations between the process parameters and the macroscopic velocity, temperature, concentration, and stress fields, and the relations between these fields and the evolution of the microstructure. In such multi-scale modelling, the transport phenomena and solid mechanics on the level of microstructure play an important role and have to be properly computationally modelled.<sup>2,3</sup> We have, in the recent years, developed a completely new generation of meshless methods, based on local collocation with radial basis functions, for solving these models on different scales. The main advantage of the method is in their similar structure in 2 and 3D, no need for polygonisation, ease of coding, high accuracy, robustness and flexibility. The method has already been developed for modelling very complex

phenomena such as macro-segregation on the macroscopic level<sup>4</sup> as well as dendritic growth on the micro-level.<sup>5</sup> An extension of the meshless method, based on a collocation with a fundamental solution (Method of Fundamental Solutions (MFS)), for the simulation of the deformation of the multiple grains in an ideal mechanical contact is presented in the present paper. An extension of the represented method with anisotropic and plastic deformation capabilities will be used in our future thermo-mechanical calculations for the microstructure evolution in metallurgical processes.<sup>6</sup>

The main idea of MFS consists of approximating the solution of the partial differential equation by a linear combination of fundamental solutions, defined in source points. The expansion coefficients are calculated by collocation or least-squares fit of the boundary conditions. The fundamental solution is usually singular in the

source points and this is the reason why the source points are located outside the domain in the MFS. Then, the original problem is reduced to determining the unknown coefficients of the fundamental solutions and the coordinates of the source points by requiring the approximation to satisfy the boundary conditions and hence solving a non-linear problem. If the source points are a priori fixed, then the coefficients of the MFS approximation are determined by solving a linear problem. The MFS has become very popular in recent years because of its simplicity. Clearly, it is applicable when the fundamental solution of the partial differential operator of the governing equation (or of the system of governing equations) of the problem under consideration is known. The MFS has been widely used<sup>7</sup> for the solution of problems in linear elasticity.

In the traditional MFS, the fictitious boundary, positioned outside the problem domain, is required to place the source points. This is very impractical or even impossible, particularly when solving multi-body problems. In recent years, various efforts have been made, aiming to remove this barrier in the MFS, so that the source points can be placed directly on the real boundary.<sup>8-12</sup> In the present paper, we use a non-singular MFS based on<sup>8</sup> to deal with the 2D multi-body isotropic elasticity problems. The application of a non-singular method of fundamental solutions (NMFS) in two-dimensional isotropic linear elasticity has been originally developed in.<sup>13</sup> We extend these developments to multi-body problems in the present paper. We respectively use area-distributed sources covering the source points to replace the concentrated point sources. This NMFS approach also does not require information about the neighbouring points for each source point, thus it is a truly mesh-free boundary method. The derivatives of the fundamental solution in the distributed source points are calculated by adopting the methodology in<sup>9</sup> from the Laplace to Kelvin fundamental solution.

The rest of the paper is structured as follows. The solution procedure is given for MFS and NMFS. Numerical results with 1, 4 and 9 bodies in contact are given, followed by conclusions and further research.

**2 GOVERNING EQUATIONS OF ELASTICITY FOR THE MULTI-BODY PROBLEM**

We consider a two-dimensional domain  $\Omega$  with the boundary  $\Gamma$ , divided into  $M$  sub-domains  $\Omega = \Omega_I \cup \Omega_{II} \cup \dots \cup \Omega_M$  with boundaries  $\Gamma = (\Gamma_I \cup \Gamma_{II} \cup \dots \cup \Gamma_M) - \Gamma_{I-II} - \dots - \Gamma_{I-M} - \dots - \Gamma_{(M-I)-M}$  as shown in **Figure 1**. Each of the domains is occupied by an isotropic, ideally elastic material with different material properties, in general. Let us introduce a two-dimensional Cartesian coordinate system with orthonormal base vectors  $\mathbf{i}_x$  and  $\mathbf{i}_y$ , and coordinates  $p_x$  and  $p_y$  of point  $P$  with the position vector  $\mathbf{p} = p_x \mathbf{i}_x + p_y \mathbf{i}_y$ . The solid is governed by Navier's equations for plane strain problems, which are the conditions for

equilibrium, expressed with the displacement  $\mathbf{u}$ . The following governing equations are valid in the sub-domain  $\Omega_m, m = I, II, \dots, M, \mathbf{p} \in \Omega_m$ :

$$\frac{2(1-\nu_m)}{1-2\nu_m} \frac{\partial^2 u_x(\mathbf{p})}{\partial p_x^2} + \frac{\partial^2 u_x(\mathbf{p})}{\partial p_y^2} + \frac{1}{1-2\nu_m} \frac{\partial^2 u_y(\mathbf{p})}{\partial p_x \partial p_y} = 0$$

$$\frac{2(1-\nu_m)}{1-2\nu_m} \frac{\partial^2 u_y(\mathbf{p})}{\partial p_y^2} + \frac{\partial^2 u_y(\mathbf{p})}{\partial p_x^2} + \frac{1}{1-2\nu_m} \frac{\partial^2 u_x(\mathbf{p})}{\partial p_x \partial p_y} = 0$$

where  $\nu_m$  represents the Poisson ratio in the subdomain  $\Omega_m$ . The boundary  $\Gamma$  is divided into two not necessarily connected parts  $\Gamma = \Gamma^D + \Gamma^T$ . On the part  $\Gamma^D$  the displacement (Dirichlet) boundary conditions are given, and on the part  $\Gamma^T$  the traction (Neumann) boundary conditions are given:

$$\chi_I u_{\zeta I}(\mathbf{p}) + \chi_{II} u_{\zeta II}(\mathbf{p}) + \dots + \chi_M u_{\zeta M}(\mathbf{p}) = \bar{u}_{\zeta}(\mathbf{p})$$

$$\zeta = x, y, \mathbf{p} \in \Gamma^D$$

$$\chi_I t_{\zeta I}(\mathbf{p}) + \chi_{II} t_{\zeta II}(\mathbf{p}) + \dots + \chi_M t_{\zeta M}(\mathbf{p}) = \bar{t}_{\zeta}(\mathbf{p})$$

$$\zeta = x, y, \mathbf{p} \in \Gamma^T$$

where:

$$\chi_m = \begin{cases} 1, & \mathbf{p} \in \Gamma_m \\ 0, & \mathbf{p} \notin \Gamma_m \end{cases}$$

On the interface between different regions, displacement continuity and traction equilibrium conditions have been assumed:

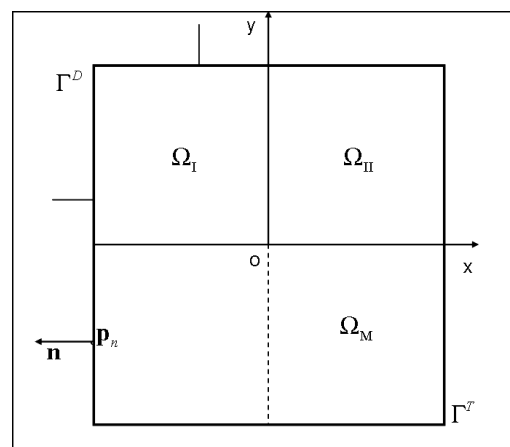
$$u_{\zeta m}(\mathbf{p}) - u_{\zeta k}(\mathbf{p}) = 0 \quad \zeta = x, y, \mathbf{p} \in \Gamma_m \cap \Gamma_k$$

$$t_{\zeta m}(\mathbf{p}) + t_{\zeta k}(\mathbf{p}) = 0 \quad \zeta = x, y, \mathbf{p} \in \Gamma_m \cap \Gamma_k$$

$$m, k = I, II, \dots, M$$

The strains  $\varepsilon_{\zeta\xi}$ ;  $\zeta, \xi = x, y$  are related to the displacement gradients by:

$$\varepsilon_{\zeta\xi} = \frac{1}{2} \left( \frac{\partial u_{\zeta}}{\partial p_{\xi}} + \frac{\partial u_{\xi}}{\partial p_{\zeta}} \right)$$



**Figure 1:** A scheme of the multi-region problem. Each of the sub-domains can have different elastic properties.

**Slika 1:** Shema problema z več območji. Vsako podobmočje ima lahko različne elastične lastnosti.

The stress components  $\sigma_{\zeta\xi}$ ;  $\zeta, \xi = x, y$  are for the plane-strain cases related to the strains through Hooke's law:

$$\sigma_{\zeta\xi} = \lambda_m \delta_{\zeta\xi} (\varepsilon_{xx} + \varepsilon_{yy}) + 2\mu_m \varepsilon_{\zeta\xi} \quad (6)$$

where  $\mu_m = E_m / 2(1 + \nu_m)$  is the shear modulus of elasticity,  $E_m$  is a constant, known as the modulus of elasticity, or Young's modulus,  $\lambda_m = 2 \nu_m \mu_m / (1 - 2\nu_m)$  is the Lamé constant, and  $\delta_{\zeta\xi}$  is the Kronecker delta:

$$\delta_{\zeta\xi} = \begin{cases} 1, & \zeta = \xi \\ 0, & \zeta \neq \xi \end{cases} \quad (7)$$

### 3 SOLUTION PROCEDURE

The fields on each of the sub-domains are represented by collocation with fundamental solutions in the boundary points. The collocation needs to satisfy the boundary conditions between different regions and outer boundaries. In a numerical implementation of MFS and NMFS, we assume that one boundary collocation point belongs to only two regions at once. In order to keep the formulation simple we do not put the discretisation points on the corners that might belong to three or more regions at once. Explicit expressions for Kelvin's fundamental solution of elastostatics, used in the collocation, are given<sup>14</sup> in a two-dimensional plane-strain situation by:

$$U_{\zeta\xi}(\mathbf{p}, \mathbf{s}) = \frac{1}{8\pi\mu(1-\nu)} \left\{ (3-4\nu) \lg\left(\frac{1}{r}\right) \delta_{\zeta\xi} + \frac{(p_\zeta - s_\zeta)(p_\xi - s_\xi)}{r^2} \right\} \quad (8)$$

$\zeta, \xi = x, y$

where the material properties depend on the position in a subdomain.  $U_{\zeta\xi}(\mathbf{p}, \mathbf{s})$  represents the displacement in the  $\zeta$  direction at point  $\mathbf{p}$  due to a unit point force acting in the  $\xi$  direction at point  $\mathbf{s}$ .  $r = \sqrt{(p_x - s_x)^2 + (p_y - s_y)^2}$  is the distance between the collocation point  $\mathbf{p}$  and the source point  $\mathbf{s}$ .

It can be shown that the following  $u_x$  and  $u_y$  satisfy the governing Eq. (1):

$$\begin{aligned} u_x(\mathbf{p}) &= \sum_{n=1}^N \tilde{U}_{xx}(\mathbf{p}, \mathbf{p}_n) \alpha_n + \sum_{n=1}^N \tilde{U}_{xy}(\mathbf{p}, \mathbf{p}_n) \beta_n \\ u_y(\mathbf{p}) &= \sum_{n=1}^N \tilde{U}_{yx}(\mathbf{p}, \mathbf{p}_n) \alpha_n + \sum_{n=1}^N \tilde{U}_{yy}(\mathbf{p}, \mathbf{p}_n) \beta_n \end{aligned} \quad (9)$$

$$\mathbf{p} \notin \sum_{n=1}^N A(\mathbf{p}_n, R)$$

where  $\alpha_n$  and  $\beta_n$  represent arbitrary constants and  $\alpha_n = \alpha_{nm}, \beta_n = \beta_{nm}, N = N_m$  when  $\mathbf{p} \in \Omega_m$ .  $N_m$  is the number of  $\mathbf{p} \in \Gamma_m, m = I, II, \dots, M$ .  $A(\mathbf{p}_n, R)$  (Figure 2) represents a circle with radius  $R$ , centred around  $\mathbf{p}_n \cdot \mathbf{p}_n = \mathbf{s}$  represents points on the physical boundary.  $\mathbf{p}$  and  $\mathbf{p}_n$  belong to the same sub-domain. The quantity  $U_{\zeta\xi}(\mathbf{p}, \mathbf{p}_n)$  is singular when  $\mathbf{p} = \mathbf{p}_n$ . We use the following de-singularization technique, proposed by<sup>6</sup>, for evaluating the singular values:

$$\tilde{U}_{\zeta\xi}(\mathbf{p}, \mathbf{p}_n) = \begin{cases} U_{\zeta\xi}(\mathbf{p}, \mathbf{p}_n) & \mathbf{p} \neq \mathbf{p}_n \\ \frac{1}{\pi R^2} \int_{A(\mathbf{s}, R)} U_{\zeta\xi}(\mathbf{p}, \mathbf{p}_n) dA & \mathbf{p} = \mathbf{p}_n \end{cases} \quad (10)$$

The tractions can be expressed as:

$$\begin{aligned} t_x(\mathbf{p}) &= \sum_{n=1}^N T_{xx}(\mathbf{p}, \mathbf{p}_n) \alpha_n + \sum_{n=1}^N T_{xy}(\mathbf{p}, \mathbf{p}_n) \beta_n \\ t_y(\mathbf{p}) &= \sum_{n=1}^N T_{yx}(\mathbf{p}, \mathbf{p}_n) \alpha_n + \sum_{n=1}^N T_{yy}(\mathbf{p}, \mathbf{p}_n) \beta_n \end{aligned} \quad (11)$$

where:

$$\begin{aligned} T_{xx}(\mathbf{p}, \mathbf{p}_n) &= \left[ \frac{2\mu(1-\nu)}{1-2\nu} \frac{\partial U_{xx}(\mathbf{p}, \mathbf{p}_n)}{\partial p_x} + \frac{2\mu\nu}{1-2\nu} \frac{\partial U_{yx}(\mathbf{p}, \mathbf{p}_n)}{\partial p_y} \right] n_{nx} + \\ & \left[ \mu \frac{\partial U_{xx}(\mathbf{p}, \mathbf{p}_n)}{\partial p_y} + \mu \frac{\partial U_{yx}(\mathbf{p}, \mathbf{p}_n)}{\partial p_x} \right] n_{ny} \\ T_{yy}(\mathbf{p}, \mathbf{p}_n) &= \left[ \frac{2\mu(1-\nu)}{1-2\nu} \frac{\partial U_{yy}(\mathbf{p}, \mathbf{p}_n)}{\partial p_x} + \frac{2\mu\nu}{1-2\nu} \frac{\partial U_{xy}(\mathbf{p}, \mathbf{p}_n)}{\partial p_y} \right] n_{nx} + \\ & \left[ \mu \frac{\partial U_{xy}(\mathbf{p}, \mathbf{p}_n)}{\partial p_y} + \mu \frac{\partial U_{yy}(\mathbf{p}, \mathbf{p}_n)}{\partial p_x} \right] n_{ny} \\ T_{yx}(\mathbf{p}, \mathbf{p}_n) &= \left[ \mu \frac{\partial U_{yx}(\mathbf{p}, \mathbf{p}_n)}{\partial p_x} + \mu \frac{\partial U_{xx}(\mathbf{p}, \mathbf{p}_n)}{\partial p_y} \right] n_{nx} + \\ & \left[ \frac{2\mu(1-\nu)}{1-2\nu} \frac{\partial U_{yx}(\mathbf{p}, \mathbf{p}_n)}{\partial p_y} + \frac{2\mu\nu}{1-2\nu} \frac{\partial U_{xx}(\mathbf{p}, \mathbf{p}_n)}{\partial p_x} \right] n_{ny} \\ T_{xy}(\mathbf{p}, \mathbf{p}_n) &= \left[ \mu \frac{\partial U_{xy}(\mathbf{p}, \mathbf{p}_n)}{\partial p_x} + \mu \frac{\partial U_{yy}(\mathbf{p}, \mathbf{p}_n)}{\partial p_y} \right] n_{nx} + \\ & \left[ \frac{2\mu(1-\nu)}{1-2\nu} \frac{\partial U_{yy}(\mathbf{p}, \mathbf{p}_n)}{\partial p_y} + \frac{2\mu\nu}{1-2\nu} \frac{\partial U_{xy}(\mathbf{p}, \mathbf{p}_n)}{\partial p_x} \right] n_{ny} \end{aligned} \quad (12)$$

$t_\zeta = t_{\zeta m}, T_{\zeta\xi} = T_{\zeta\xi m}, n_{n\zeta} = n_{n\zeta m}, \zeta, \xi = x, y$  when  $\mathbf{p} \in \Omega_m$ .

The coefficients  $\alpha_n$  and  $\beta_n$  are calculated from a system of  $(2N_I + 2N_{II} + \dots + 2N_M) \times (2N_I + 2N_{II} + \dots + 2N_M)$  algebraic equations, obtained by collocating the boundary conditions:

$$\mathbf{Ax} = \mathbf{b} \quad (13)$$

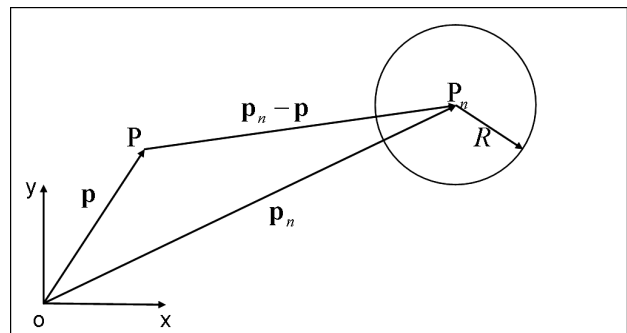


Figure 2: Distributed source on a disk  $A(\mathbf{p}_n)$   
Slika 2: Porazdeljeni izviri na disku  $A(\mathbf{p}_n)$

where  $\mathbf{A}$  is composed of  $\tilde{U}_{\zeta\xi}(\mathbf{p}, \mathbf{p}_n)$  and  $\tilde{T}_{\zeta\xi}(\mathbf{p}, \mathbf{p}_n)$ ,  $\mathbf{x}$  is composed of  $\alpha_n$  and  $\beta_n$ , and  $\mathbf{b}$  is composed of  $u_\zeta$ ,  $t_\zeta$  and 0. The explicit form of the elements of the algebraic equation system (13) can be found in.<sup>11</sup>

The diagonal terms  $\tilde{T}_{\zeta\xi}(\mathbf{p}_l, \mathbf{p}_l)$ ,  $\zeta, \xi = x, y, l = 1, \dots, N_I + \dots + N_M$ , in Eq. (13) are determined indirectly for the collocation points on  $\Gamma^T$ . For this purpose, the method proposed in<sup>7</sup> for potential problems is applied to determine the diagonal coefficients of Eq. (13). In the approach, we first assume two simple solutions. The first simple solution is  $\bar{u}_x(\mathbf{p}) = p_x + c_x$ ,  $\bar{u}_y(\mathbf{p}) = 0$  everywhere. The second simple solution is  $\bar{u}_x(\mathbf{p}) = 0$ ,  $\bar{u}_y(\mathbf{p}) = p_y + c_y$  everywhere. We solve them for the corresponding  $\alpha_n^{(1)}$ ,  $\beta_n^{(1)}$  and  $\alpha_n^{(2)}$ ,  $\beta_n^{(2)}$  using only the displacement boundary condition. From these two solutions, we can also know:

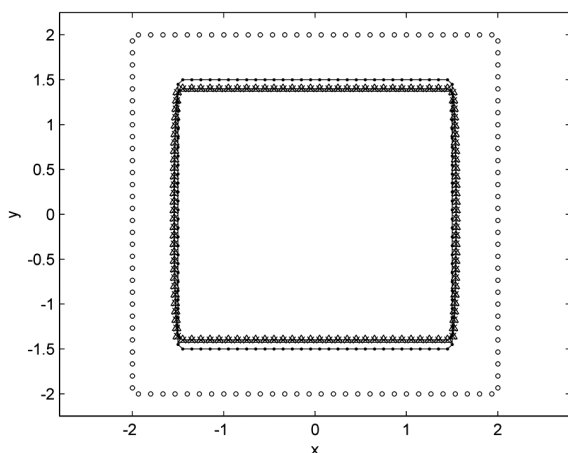
$$\frac{\partial u_x^{(1)}(\mathbf{p})}{\partial p_x} = 1, \quad \frac{\partial u_x^{(1)}(\mathbf{p})}{\partial p_y} = \frac{\partial u_y^{(1)}(\mathbf{p})}{\partial p_x} = \frac{\partial u_y^{(1)}(\mathbf{p})}{\partial p_y} = 0 \tag{14}$$

$$\frac{\partial u_x^{(2)}(\mathbf{p})}{\partial p_x} = \frac{\partial u_x^{(2)}(\mathbf{p})}{\partial p_y} = \frac{\partial u_y^{(2)}(\mathbf{p})}{\partial p_x} = 0, \quad \frac{\partial u_y^{(2)}(\mathbf{p})}{\partial p_y} = 1$$

By substituting  $\alpha_n^{(1)}$ ,  $\beta_n^{(1)}$  and  $\alpha_n^{(2)}$ ,  $\beta_n^{(2)}$  and Eq. (9) into Eq. (14), we can obtain the diagonal terms  $\tilde{T}_{\zeta\xi}(\mathbf{p}_l, \mathbf{p}_l)$ ,  $\zeta, \xi = x, y, l = 1, \dots, N_I + \dots + N_M$ . The constant  $c$  should be selected in such a way that all the points in the upper cases do not move the same distance. So that the denominators in the upper derivations are not zero.

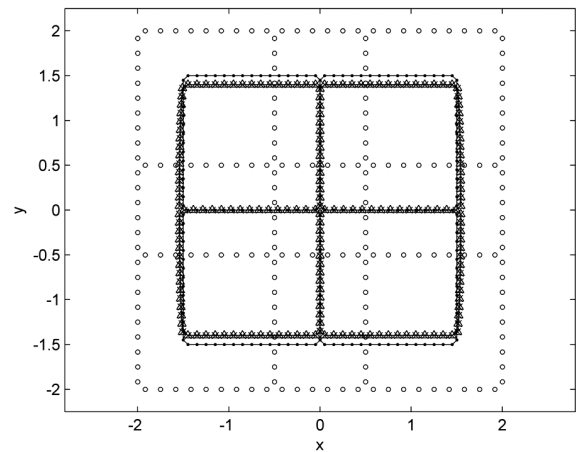
By knowing all the elements  $A_{ij}$  and  $b_k$  of the system (13), we can determine all the values of  $\alpha_n$  and  $\beta_n$ . Subsequently, we can calculate the displacement for all the domain points using Eq. (9).

### 4 NUMERICAL EXAMPLES



**Figure 3:** The deformation calculated with MFS and NMFS for a one-domain case with  $E = 1\text{N/m}^2$ ,  $\nu = 0.3$  and  $N = 120$  (●: collocation points, ○: source points, x: MFS solution, Δ: NMFS solution)

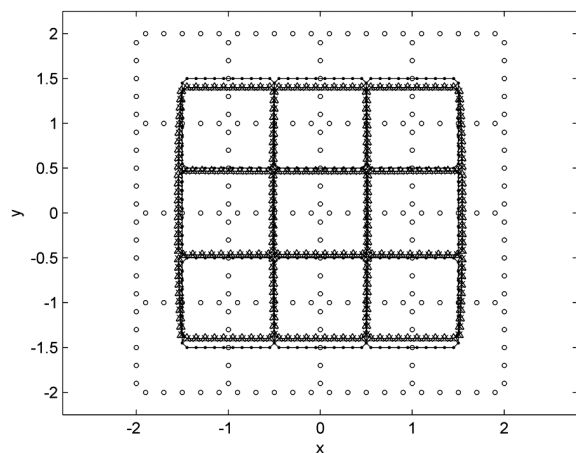
**Slika 3:** Deformacija, izračunana z MFR in NMFR, za primer z enim območjem z  $E = 1\text{N/m}^2$ ,  $\nu = 0,3$  in  $N = 120$  (●: kolokacijska točka, ○: izvorna točka, x: MFR-rešitev, Δ: NMFR-rešitev)



**Figure 4:** The deformation calculated with MFS and NMFS for a four-domain case  $E_I = E_{II} = \dots = E_{IV} = 1\text{N/m}^2$ ,  $\nu_I = \nu_{II} = \dots = \nu_{IV} = 0.3$  and  $N = 240$  (●: collocation points, ○: source points, x: MFS solution, Δ: NMFS solution). The position of the source points in MFS is on squares around each of the square physical domains.

**Slika 4:** Deformacija, izračunana z MFR in NMFR, za primer s štirimi območji  $E_I = E_{II} = \dots = E_{IV} = 1\text{N/m}^2$ ,  $\nu_I = \nu_{II} = \dots = \nu_{IV} = 0,3$  in  $N = 240$  (●: kolokacijska točka, ○: izvorna točka, x: MFR-rešitev, Δ: NMFR-rešitev). Pozicija izvornih točk je na kvadratih okoli vsakega od štirih kvadratnih domen.

We consider a square with side  $a = 3\text{m}$  centred around  $p_x = 0\text{m}$ ,  $p_y = 0\text{m}$  for testing the performance of the method. We distinguish three sub-examples. In the first one, the whole square is occupied by one material, with the material properties  $E = 1\text{N/m}^2$ ,  $\nu = 0.3$ , in the second one, the square is split into four parts with the same material properties as in the first example  $E_I = E_{II} = \dots = E_{IV} = 1\text{N/m}^2$ ,  $\nu_I = \nu_{II} = \dots = \nu_{IV} = 0.3$ , and in the third one, the square is split into nine parts with the same



**Figure 5:** The deformation calculated with MFS and NMFS for a nine-domain case  $E_I = E_{II} = \dots = E_{IV} = 1\text{N/m}^2$ ,  $\nu_I = \nu_{II} = \dots = \nu_{IV} = 0.3$  and  $N = 360$  (●: collocation points, ○: source points, x: MFS solution, Δ: NMFS solution). The position of the source points in MFS is on squares around each of the square physical domains.

**Slika 5:** Deformacija, izračunana z MFR in NMFR, za primer z devetimi območji  $E_I = E_{II} = \dots = E_{IV} = 1\text{N/m}^2$ ,  $\nu_I = \nu_{II} = \dots = \nu_{IV} = 0,3$  in  $N = 360$  (●: kolokacijske točke, ○: izvorne točke, x: MFR-rešitev, Δ: NMFR-rešitev). Pozicija izvornih točk je na kvadratih okoli vsakega od štirih kvadratnih domen.

material properties as in the first example  $E_I = E_{II} = \dots = E_{IX} = 1\text{N/m}^2$   $\nu_I = \nu_{II} = \dots = \nu_{IX} = 0.3$ . We considered the solution of the Navier equations in this square subject to the boundary conditions  $\bar{u}_x = 0\text{m}$ ,  $\bar{u}_y = -0.1\text{m}$  on the points of the north side with  $p_y = 1.5\text{m}$ ;  $\bar{u}_x = 0\text{m}$ ,  $\bar{u}_y = 0.1\text{m}$  on the south side with  $p_y = -1.5\text{m}$  and on the east  $p_x = 1.5\text{m}$  and west  $p_x = -1.5\text{m}$  sides  $\bar{t}_x = 0\text{N/m}^2$ ,  $\bar{t}_y = 0\text{N/m}^2$  is set. A plot of the deformation, calculated with the defined three sub-examples is shown in **Figures 3, 4 and 5**, respectively. The following parameters were used  $R = d/5$ ,  $R_I = d_I/5$ , ...,  $R_{IX} = d_{IX}/5$ , where  $d$ ,  $d_m$ ,  $m = \text{I, II, ..., IX}$  are the smallest distances between two nodes on the boundary,  $R_m$  is the radius of the circle centred the point  $\mathbf{p}_n \in \Gamma_m$ ,  $c_x = c_y = c_{xI} = c_{yI} = L = c_{xIX} = c_{yIX} = 4$ . The distance of the fictitious boundary from the true boundary for the MFS is set to  $R_M = 5d$ ,  $R_{MI} = 5d_I$ , ...,  $R_{MIX} = 5d_{IX}$ .

**Figures 3, 4 and 5** show good agreement between the solution for a one-domain region and a solution recalculated with the four and nine regions in ideal mechanical contact and with the same material properties. The maximum absolute difference in displacements between the values in **Figures 3 and 4** at the outer boundary are  $\Delta u_x = 0.0417\text{m}$ ,  $\Delta u_y = 0.0886\text{m}$ , and between **Figures 3 and 5**  $\Delta u_x = 0.0017\text{m}$ ,  $\Delta u_y = 0.0012\text{m}$ , respectively.

## 5 CONCLUSION

A new, non-singular method of fundamental solutions<sup>13</sup> is extended in the present paper to solve multi-dimensional linear elasticity problems. In this approach, the singular values of the fundamental solution are integrated over small circular discs, so that the coefficients in the system of equations can be evaluated analytically and consistently, leading to an extremely simple computer implementation of this method. The method essentially gives the same results as the classic MFS. It has the advantage that the artificial boundary is not present; however, at the expense of solving three times the systems of algebraic equations in comparison with only one solution in MFS. The main advantage of the method is that the discretisation is performed only on the boundary of the domain and no polygonisation is needed, like in the finite-element method. The NMFS method, presented in this paper, can be adapted or extended to handle many related problems, such as three-dimensional elasticity, anisotropic elasticity, and multi-body problems, which all represent directions for

our further investigations. The advantage of not having to generate the artificial boundary is particularly welcome in these types of problems. The method will be used in the future for the calculation of multigrain deformation problems in steel and aluminium alloys, with realistic grain shapes, obtained from the microscope images. The developed method is believed to represent a most simple state-of-the-art way to numerically cope with these types of problems.

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## 6 REFERENCES

- 1 B. Šarler, R. Vertnik, S. Šaletič, G. Manojlović, J. Cesar, Berg-Huettenmaenn. Monatsh., 150 (2005), 300–306
- 2 T. Mura, Micromechanics of Defects in Solids, 2ed, Martinus Nijhoff Publishers, Netherlands 1987
- 3 M. Braccini, M. Dupeux, Mechanics of Solid Interfaces, Wiley, New York 2012
- 4 G. Kosec, M. Založnik, B. Šarler, H. Ccombeau, CMC: Computers, Materials & Continua, 22 (2011), 169–195
- 5 A. Z. Lorbiecka, B. Šarler, CMC: Computers, Materials & Continua, 18 (2010), 69–103
- 6 U. Hanoglu, S. Islam, B. Šarler, Mater. Tehnol., 45 (2011) 6, 545–547
- 7 Y. S. Smyrlis, Mathematics of Computation, 78 (2009), 1399–1434
- 8 Y. J. Liu, Engineering Analysis with Boundary Elements, 34 (2010), 914–919
- 9 B. Šarler, Engineering Analysis with Boundary Elements, 33 (2009), 1374–1382
- 10 D. L. Young, K. H. Chen, J. T. Chen, J. H. Kao, CMES: Computer Modeling in Engineering & Sciences, 19 (2007), 197–222
- 11 D. L. Young, K. H. Chen, C. W. Lee, Journal of Computational Physics, 209 (2005), 290–321
- 12 W. Chen, F. Z. Wang, Engineering Analysis with Boundary Elements, 34 (2010), 530–532
- 13 Q. G. Liu, B. Šarler, CMES: Computer Modeling in Engineering and Sciences, 91 (2013), 235–266
- 14 D. E. Beskos, Boundary Element Methods in Mechanics, Elsevier, Amsterdam 1987, 33